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ANALYTICAL AND EXPERIMENTAL INVESTIGATION ON VIBRATING MEMBRANES WITH A CENTRAL POINT SUPPORT

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1. INTRODUCTION

In a recent, interesting study Wang [1] analyzed transverse vibrations of polygonal membranes with a central, fixed circular core by means of the point matching technique. He also studied in detail the asymptotic properties where the center core shrinks to a point, i.e., the membrane is point-supported at the center. Furthermore, Wang's study shows that in the case of the circular, annular membrane with a pinpoint restraint ($\varepsilon = 0$) one obtains the fundamental frequency of the solid circular membrane. Accordingly he expects this property to hold for polygonal membranes [1].

It was shown in a recent discussion that the presence of a pinpoint support at the center of a circular membrane does not alter the values of higher frequency coefficients corresponding to axisymmetric modes [2].

The present study deals with an experimental study on vibrating circular and square membranes with a central, point support complementing reference [2] by including additional numerical information on higher axisymmetric modes of a circular membrane.

2. THE VIBRATING CIRCULAR MEMBRANE WITH A CENTRAL PINPOINT SUPPORT

The classical problem of vibrating membranes is described, in the case of normal modes, by the Helmholtz equation,

$$\nabla^2 \Psi + k^2 \Psi = 0, \tag{1}$$

where k is the eigenvalue corresponding to a normal mode.

When dealing with a vibrating circular annular membrane described by the axisymmetric modes equation (1) and the prescribed boundary conditions at r = b, a, where b is the outer radius and a is the inner radius of the membrane, the solution yields the well known frequency equation

$$Y_0(k \ b) \ J_0(k \ a) - J_0(k \ b) \ Y_0(k \ a) = 0 \tag{2}$$

Defining now

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$$\varepsilon = a/b \tag{3}$$

and substituting in equation (2) one obtains

$$Y_0(\Omega)J_0(\Omega\varepsilon) - J_0(\Omega)Y_0(\Omega\varepsilon) = 0, \qquad (4)$$

where $\Omega = kb$ and where the usual notation for Bessel's function of first and second kind has been used.

As ε becomes very small in equation (4) one is able to approach, asymptotically, the case of a central, pinpoint support [2].

Values of $\Omega_i = \sqrt{\rho/S \omega_i b}$ (where ρ is the density per unit area of the membrane and S is the applied force per unit length applied at each boundary of the membrane) are shown in Table 1 for the first five modes of vibration. Also shown in the table are the exact eigenvalues corresponding to axisymmetric modes of vibration of a solid circular membrane.

The numerical determinations have been performed using MAPLE [3]. It is concluded that as ε decreases and reaches the value 10^{-1000} the differences with the exact eigenvalues corresponding to a solid, circular membrane are of the order of 0.01%. This conclusion agrees with the one reached in reference [3] for the first three modes.

3. EXPERIMENTAL INVESTIGATION

Figure 1 depicts the experimental set-up used in the case of a vibrating circular membrane.* The first three natural frequencies corresponding to axisymmetric modes were excited using a commercial speaker. A specially developed infrared transducer [4] was used to detect the frequencies which were analyzed by means

The first five frequency coefficients of a circular annular membrane as a function of $\varepsilon = a/b$ and as ε becomes very small

TABLE 1

3	$arOmega_1$	$arOmega_2$	$arOmega_3$	Ω_4	Ω_5
10^{-1} 10^{-2}	3.3139	6·8575	10.3774	13.8864	17.3896
10^{-3}	2.8009 2.6548	5·8089	9·2141 8·9676	12·4113 12·1251	15.2811
10^{-4} 10^{-5}	2.58/1 2.5482	5·7236 5.6769	8.8698 8.8181	12·0170 11·9614	15·1641 15.1052
10^{-10} 10^{-100}	2·4741 2·4115	5·5929 5·5269	8·7283 8·6605	11·8673 11·7984	15·0075 14·9378
$\frac{10^{-1000}}{\text{Exact}}$	2.4055 2.4048	5.5207 5.5200	8·6544 8·6537	11·7922 11·7915	14·9316 14·9309

Note: the values defined as "exact" correspond to axisymmetric modes of a solid, circular membrane.

*A professional Brazilian musical drum was used (2b = 150 mm).



Figure 1. Experimental set-up and test of the circular membrane.

of CSI 1900. Experiments were performed with and without a central, point support. When performing the first set of experiments, special care was placed on not introducing variations of the in-plane stress field corresponding to the membrane without the central support.

Typical experimental results are shown in Table 2. It is concluded that placing a pinpoint central support raises the fundamental frequency by about 7%, which can be considered a very small increment in view of the inherent difficulties caused by testing a real membrane. (In the case of a simply supported circular plate with a central support the fundamental frequency is practically three times larger than the one corresponding to the plate without a central support [5].) The increment is considerably smaller in the case of the second and third modes; see, Table 2.

A square membrane was constructed out of thin leather of dimensions 128×128 mm and 1.2 mm thick and tested; see Figure 2. The experimental

	membrane: effect of the central, point support			
	f_1 (Hz)	f_2 (Hz)	<i>f</i> ₃ (Hz)	
Without central support	217.5	458.8	805.5	
With central support	232.5	472	808	

TABLE 2

Values of natural frequencies determined experimentally in the case of a circular



Figure 2. Experimental set-up and test of the square membrane.

procedure followed was similar to the one used when testing the circular membrane.

Table 3 shows the first three frequencies corresponding to symmetric modes when the membrane is centrally supported and when it is only boundarysupported. One concludes, again, that the frequency variation is negligible for

TABLE	3
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Values of natural frequencies corresponding to doubly-symmetric modes in the case of a square membrane determined experimentally: effect of the central, point support

	f_1 (Hz)	f_2 (Hz)	f_3 (Hz)
Without central support	149.9	315	750.9
With central support	175.4	318.5	749.7



Figure 3. Fundamental mode shape of the annular membrane for $\varepsilon = 10^{-5}$.

the second and third frequencies and more noticeable in the case of the fundamental frequency.

Finally, Figures 3 and 4 depict the fundamental mode shapes for $\varepsilon = 10^{-5}$ and 10^{-100} , respectively, in the case of the circular annular membrane. It is observed that as ε becomes very small ($\varepsilon = 10^{-100}$), the mode shape resembles the functional relation defined by Bessel's function of the first kind and order zero, with a sudden drop at $\varepsilon = 10^{-100}$ in order to satisfy the prescribed boundary condition.

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Figure 4. Fundamental mode shape of the annular membrane for $\varepsilon = 10^{-100}$.

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REFERENCES

- 1. C. Y. WANG 1998 *Journal of Sound and Vibration* **215**, 195–199. On the polygonal membrane with a circular core.
- 2. P. A. A. LAURA and S. A. VERA 1998 Publication No. 98–13. Institute of Applied Mechanics (CONICET, Bahía Blanca, Argentina). Comments on: "On the polygonal membrane with a circular core".
- 3. B. W. CHAR, K. O. GEDDES, G. H. GONNET, B. L. LEONG, M. B. MONAGAN and S. M. WATT 1991 *Maple V. Library Reference Manual*. Berlin: Springer.
- 4. S. LA MALFA 1995 *The Shock and Vibration Digest* 27, 18–20. Infrared principles applied to the study of vibrations of mechanical systems.
- 5. A. W. LEISSA 1969 NASA SP160 Vibration of plates.